

1 **BRIDGE DECK FLUTTER DERIVATIVES: EFFICIENT NUMERICAL EVALUATION**  
2 **EXPLOITING THEIR INTERDEPENDENCE.**

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6  
7 **ABSTRACT**

8 Increasing the efficiency in the process to numerically compute the flutter derivatives of bridge  
9 deck sections is desirable to advance the application of CFD based aerodynamic design in  
10 industrial projects. In this paper, a 2D unsteady Reynolds-averaged Navier-Stokes (URANS)  
11 approach adopting Menter's SST  $k-\omega$  turbulence model is employed for computing the flutter  
12 derivatives and the static aerodynamic characteristics of two well known examples: a rectangular  
13 cylinder showing a completely reattached flow and the generic G1 section representative of  
14 streamlined deck sections. The analytical relationships between flutter derivatives reported in the  
15 literature are applied with the purpose of halving the number of required numerical simulations for  
16 computing the flutter derivatives. The solver of choice has been the open source code OpenFOAM.  
17 It has been found that the proposed methodology offers results which agree well with the  
18 experimental data and the accuracy of the estimated flutter derivatives is similar to the results  
19 reported in the literature where the complete set of numerical simulations has been performed for  
20 both heave and pitch degrees of freedom.

21

22 **KEYWORDS:** Computational fluid dynamics, bluff body aerodynamics, flutter derivatives,  
23 rectangular cylinder, streamlined deck sections.

24

25 **1. INTRODUCTION**

26 Long span bridges are prone to aeroelastic phenomena such as vortex induced vibrations, flutter or  
27 buffeting. In fact, safety against flutter instability is one of the fundamental requirements in long  
28 span bridge design. If the wind speed exceeds the critical flutter speed of the structure, self-excited  
29 oscillations of the deck would rapidly amplify causing the collapse of the bridge.

30 The most widely used method for the identification of the flutter critical wind speed is Scanlan's  
31 approach, developed in the 1970s (Scanlan and Tomko, 1971), where a set of semi-empirical  
32 functions, named flutter derivatives, must be identified in order to define the motion-induced  
33 aerodynamic load acting on the bridge deck (Bartoli and Mannini, 2008). Traditionally, the  
34 identification of flutter derivatives has been conducted by means of wind tunnel tests of sectional  
35 models of bridge decks. The application in recent years of numerical methods in the identification  
36 of flutter derivatives aims at avoiding expensive and cumbersome experimental campaigns which  
37 are the standard approach in industrial applications currently.

38 In Computational Fluid Dynamics (CFD) modeling the flutter derivatives identification can be  
39 done following two different approaches (Fransos and Bruno, 2006). The first one requires the  
40 simulation of the forced harmonic oscillations in pitch and heave degrees of freedom. Then, the  
41 flutter derivatives are identified from the amplitude and phase relationships between the imposed  
42 displacement and the induced aeroelastic forces. The second method, based on indicial theory,

43 requires simulating an abrupt displacement of the body immersed in the flow, which causes non-  
44 stationary forces. The flutter derivatives can then be computed from the ratio between the Fourier  
45 transforms of the step-response non-stationary forces and the prescribed step-input displacement.  
46 The methodology, based on the simulation of forced oscillations, has been, by far, more widely  
47 used than the one based on the indicial approach despite the apparent efficiency of the indicial  
48 function approach.

49 Focusing on applications of the harmonic forced oscillations approach, the trend in the 1990's and  
50 early 2000's has been developing in-house CFD solvers based on the finite-difference, finite  
51 element, finite volume or discrete vortex methods. The references in the literature are numerous  
52 and some examples, without intending to be exhaustive are: Mendes and Branco (1998), Larsen  
53 and Walther (1998), Morgenthal and McRobie (2002), Xiang and Ge (2002), Vairo (2003), Jeong  
54 and Kwon (2003), Frandsen (2004), Zhu et al. (2007) and Zhu et al. (2009). Developing in-house  
55 software has obviously been a barrier for the application of numerical methods in industrial bridge  
56 design problems due to its scientific complexity and the required labor and financial resources.  
57 Therefore more recently the focus has been put on applying general purpose commercial finite  
58 volume solvers in bridge aerodynamics problems. An early application was authored by Bruno et  
59 al. (2001) who used FLUENT for studying the aerodynamic response of a static box deck and the  
60 effect of section details such as fairings and barriers. Fluid-structure interaction problems have  
61 been addressed more recently. In Ge and Xiang (2008) both in-house solvers and the commercial  
62 code FLUENT are applied, depending on the chosen approach for turbulence modeling. Sarwar et  
63 al. (2008) obtained the flutter derivatives of a bridge deck section and high aspect ratio rectangular  
64 cylinders by means of 3D Large Eddy Simulation (LES) using FLUENT. Huang et al. (2009) also  
65 used FLUENT to compute the flutter derivatives of the Great Belt Bridge and the Sutong Yangtze  
66 cable-stayed bridge. Starossek et al. (2009) employed the commercial software COMET to obtain  
67 the flutter derivatives of 31 different bridge sections, including experimental validation for a subset  
68 of 9 sections tested in a water tunnel. Bai et al. (2010) used a combination of in-house code and  
69 ANSYS-CFX commercial software for computing force coefficients and flutter derivatives of  
70 various 3D deck sections. Huang and Liao (2011) used FLUENT to simulate forced oscillations of  
71 a flat plate and a bridge deck containing a linear combination of a set of frequencies. Also, Brusiani  
72 et al. (2013) employed FLUENT to compute the flutter derivatives of the Great Belt Bridge using a  
73 different turbulence model than Huang and co-workers. Of particular interest is the growing use of  
74 open source general CFD solvers. In Sarkic et al. (2012), the open source code OpenFOAM is  
75 applied to numerically replicate the wind tunnel test for identifying the force coefficients and flutter  
76 derivatives of a box deck cross-section. A more recent application by some of the authors of the  
77 former reference can be found in Sarkic and Höffer (2013) where the LES turbulence model is  
78 applied to the same box deck.

79 CFD applications based on indicial functions are scarce in spite of its potential. In Bruno and  
80 Fransos (2008) it has been remarked that in this method just a single simulation for each degree of  
81 freedom is required to identify the complete set of flutter derivatives and that only the transient  
82 flow needs to be simulated. Thus, this approach is less demanding in computational resources than  
83 the classical forced oscillation based method. On the other hand, the problem is particularly  
84 challenging from the CFD simulation perspective. Early applications are Lesieutre et al. (1994)  
85 who simulated the motion of a wing in the frame of an application to aircraft manoeuvres and Brar  
86 et al. (1996) who applied the Finite Element Method to obtain the flutter derivatives of an airfoil  
87 and a rectangular cylinder. A modified smoothed indicial approach was further developed in  
88 Fransos and Bruno (2006) and Bruno and Fransos (2008) who used FLUENT to obtain the flutter

89 derivatives of a flat plate of finite thickness and studied also the effect of the Reynolds number on  
90 the flutter derivatives. The indicial approach has also been applied in the frame of a probabilistic  
91 study of the aerodynamic and aeroelastic responses of a flat plate (Bruno et al, 2009). More  
92 recently Zhu and Gu (2014) have presented a method to extract the flutter derivatives of  
93 streamlined bridge decks, even if the application of the modified indicial approach to bluff bodies  
94 remains questionable.

95 From the previous review of the state of the art regarding applications of CFD in the design of long  
96 span bridges, the main reasons why numerical simulations are not being generally applied in bridge  
97 design in the industry to complement wind tunnel tests need to be discussed. Developing and  
98 upgrading in-house software is a complex task and requires highly skilled personnel and substantial  
99 funding. Consequently it can only be achieved by a small number of organizations in the world.  
100 The increasing use of commercial software in recent years is making it easier to access the required  
101 technology. However, the cost of licenses, particularly for running massively parallel simulations,  
102 in many cases prevents the extensive use of CFD in design problems. This circumstance has made  
103 particularly appealing the use of open source solvers for both industry and academia, and open  
104 source software has already been applied in bridge design problems. Besides this, the increasing  
105 number of published successful simulations in bridge related problems means that CFD techniques  
106 are nowadays more mature and therefore more robust and reliable.

107 In spite of the dramatic improvements in computational power and access to cluster technology of  
108 recent years, the computer power demands linked with modeling complex fluid-structure  
109 interaction problems remains a key issue. In this respect, any method or technique which allows  
110 decreasing computational demands would facilitate incorporating CFD based design in bridge  
111 engineering design. A number of researchers have proposed explicit relationships between flutter  
112 derivatives which have proved to be reliable for streamlined bridge decks such as Matsumoto  
113 (1996), Scanlan et al. (1997), Chen and Kareem (2002) or Tubino (2005). The application of these  
114 formulae allows the number of computer simulations for obtaining the flutter derivatives to be  
115 reduced to just half of the number required following the standard approach based on forced  
116 harmonic vibrations in heave and pitch degrees of freedom. To the authors' knowledge the  
117 aforementioned approach has not been applied in CFD-based studies to date.

118 The aim of the current piece of research is to propose a cost effective, and therefore efficient,  
119 computer based approach for obtaining force coefficients and flutter derivatives of bridge deck box  
120 sections which could be used in industrial applications where the shape of different bridge deck  
121 designs could be numerically optimized. Consequently, a 2D URANS strategy is proposed, using  
122 the general purpose open source CFD solver OpenFOAM v2.1.1 in combination with the explicit  
123 relationships between flutter derivatives mentioned above. The more demanding 3D Detached  
124 Eddy Simulation (DES) or LES approaches, in spite of their superior accuracy, have not been  
125 considered in this work since they would pose additional challenges in terms of higher computer  
126 power demands and model setup.

127 A rectangular cylinder showing a separated and reattached time-averaged flow pattern has been  
128 selected as one of the case studies for the computation of the flutter derivatives. In particular, a  
129 ratio  $B/H=4.9$  rectangular cylinder ( $B$  is the prism width and  $H$  is the height) was chosen in order to  
130 replicate an existing sectional model at the wind tunnel of the University of Nottingham. In the  
131 literature, the number of published references, both experimental and computational, dealing with  
132 the response of  $B/H=5$  rectangular cylinders is plentiful, to a great extent thanks to the BARC  
133 initiative (Bruno et al. 2014). Taking into account the expected minimal differences between the

134 aerodynamic response of  $B/H=4.9$  and  $B/H=5$  rectangular cylinders, for the sake of the efficiency  
 135 of means in research, the authors have considered that the existing literature on 5:1 rectangular  
 136 cylinders is adequate for the validation of the force coefficients and the flutter derivatives of the  
 137  $B/H=4.9$  rectangular cylinder at  $0^\circ$  angle of attack. However, in the case that additional numerical  
 138 studies would require validation against experimental data outside the range found in the literature,  
 139 further wind tunnel tests could readily be conducted using the existing  $B/H=4.9$  sectional model.

140 The second application case has been the G1 generic box section described in Scanlan and Tomko  
 141 (1971) and Larsen and Walther (1998). The modern practice in long span bridge design has  
 142 incorporated box deck cross-sections as the most common choice for these challenging structures.  
 143 There are several reasons for this: a good aerodynamic and aeroelastic response characteristic of  
 144 streamlined cross-sections, high torsional stiffness, construction economy and, in many cases,  
 145 superior aesthetic value compared to truss girders. Recent examples of applications comprising box  
 146 decks are the Forth Replacement Crossing in the United Kingdom, the Normandy Bridge and  
 147 Millau Viaduct, in France, the Sutong Bridge in China or the Russky Bridge in Russia, amongst  
 148 many others.

149 In the first part of this paper, the fundamental formulation and the numerical approach adopted,  
 150 along with the computational models, for simulating the aerodynamic response of the bridge decks  
 151 are explained. Then, the results of the study of the sensitivity of the solution to the spatial and  
 152 temporal discretisations for the G1 generic section are summarized. Next the aerodynamic  
 153 characteristics of the static  $B/H=4.9$  rectangular cylinder are analyzed based on the values of the  
 154 Strouhal number, force coefficients and the distribution of the averaged pressure coefficient and its  
 155 standard deviation. Then the flutter derivatives of the rectangular cylinder, where the relationships  
 156 between flutter derivatives have been applied, are reported and compared with wind tunnel data. It  
 157 follows the analysis of the characteristics of the static G1 section based on force coefficients and  
 158 the distribution of the averaged pressure coefficient. The results section ends with the report of the  
 159 flutter derivatives of the G1 section and the corresponding comparison with experimental and other  
 160 numerical data in the literature. Finally, conclusions are drawn from the work reported herein.

## 161 2. NUMERICAL FORMULATION

162 The flow around the bluff bodies of interest is modeled by means of the unsteady Reynolds-  
 163 averaged Navier-Stokes equations considering incompressible flow. A 2D URANS approach has  
 164 been preferred which, according to Brusiani et al. (2013), is equivalent to imposing the perfect  
 165 correlation of the flow structures in the span-wise direction.

166 The time averaging of the equations for conservation of mass and momentum gives the Reynolds  
 167 averaged equations of motion in conservative form. According to Wilcox (2006):

168

$$\frac{\partial U_i}{\partial x_i} = 0 \quad (1.a)$$

$$\rho \frac{\partial U_i}{\partial t} + \rho U_j \frac{\partial U_i}{\partial x_j} = - \frac{\partial P}{\partial x_i} + \frac{\partial}{\partial x_j} (2\mu S_{ij} - \rho \overline{u'_i u'_j}) \quad (1.b)$$

169

170 where  $U_i$  is the mean velocity vector,  $x_i$  is the position vector,  $t$  is the time,  $\rho$  is the fluid density,  
 171 assumed constant,  $u'_i$  is the fluctuating velocity and the overbar represents the time average,  $P$  is

172 the mean pressure,  $\mu$  is the fluid viscosity and  $S_{ij}$  is the mean strain-rate tensor. From the above  
 173 equation, the specific Reynolds stress tensor is defined as:

$$\tau_{ij} = -\overline{u_i u_j} \quad (2)$$

174

175 which is an additional unknown to be modeled based on the Boussinesq assumption for one and  
 176 two equation turbulence models (Wilcox, 2006).

$$\tau_{ij} = 2\nu_T S_{ij} - \frac{2}{3}k\delta_{ij} \quad (3)$$

177

178 where  $\nu_T$  is the kinematic eddy viscosity,  $S_{ij}$  is the mean strain-rate tensor and  $k$  is the turbulent  
 179 kinetic energy per unit mass.

180 In this work the closure problem is solved applying Menter's  $k-\omega$  SST model for incompressible  
 181 flows, reported in Menter and Esch (2001).

182 For the simulations where forced oscillations of the bluff body have been imposed, the Arbitrary  
 183 Lagrangian Eulerian (ALE) formulation has been applied for allowing movements of the mesh  
 184 inside the computational domain. The conservation of mass and momentum equations are written  
 185 as follows (Bai et al., 2010, Sarkic et al., 2012):

$$\frac{\partial(U_i - U_{gi})}{\partial x_i} = 0 \quad (4.a)$$

$$\rho \frac{\partial U_i}{\partial t} + \rho U_j \frac{\partial(U_i - U_{gi})}{\partial x_j} = -\frac{\partial P}{\partial x_i} + \frac{\partial}{\partial x_j} (2\mu S_{ij} - \rho \overline{u_i u_j}) \quad (4.b)$$

186

187 where  $U_{gi}$  is the grid velocity in the  $i$ -th direction.

### 188 3. FORCE COEFFICIENTS AND FLUTTER DERIVATIVES COMPUTATION BY 189 MEANS OF FORCED OSCILLATION SIMULATIONS

190 The definition of the force coefficients considered in this study is given in (5):

$$C_d = \frac{D}{\frac{1}{2}\rho U^2 B} \quad C_l = \frac{L}{\frac{1}{2}\rho U^2 B} \quad C_m = \frac{M}{\frac{1}{2}\rho U^2 B^2} \quad (5)$$

191

192 In the former expressions  $D$  is the drag force per span length, positive in the windward direction,  $L$   
 193 is the lift force per span length, positive upwards, and  $M$  is the pitching moment per span length,  
 194 positive clockwise,  $\rho$  is the fluid density,  $U$  is the flow speed and  $B$  is the bluff body width.

195 Flutter derivatives are semi-empirical parameters which relate motion-induced forces ~~and~~ with the  
 196 displacements of the structure and their time derivative. These parameters have traditionally been  
 197 identified using wind tunnel tests, but more recently, numerical based simulations have been  
 198 applied.

199 According to Simiu and Scanlan (1996), the aeroelastic forces on a bridge deck, considering two  
 200 degrees of freedom (heave and pitch), can be written as follows:

$$L_{ae} = \frac{1}{2}\rho U^2 B \left[ KH_1^* \frac{\dot{h}}{U} + KH_2^* \frac{B\dot{\alpha}}{U} + K^2 H_3^* \alpha + K^2 H_4^* \frac{h}{B} \right] \quad (6.a)$$

$$M_{ae} = \frac{1}{2}\rho U^2 B^2 \left[ KA_1^* \frac{\dot{h}}{U} + KA_2^* \frac{B\dot{\alpha}}{U} + K^2 A_3^* \alpha + K^2 A_4^* \frac{h}{B} \right] \quad (6.b)$$

201

202 where  $L_{ae}$  is the aeroelastic force per unit of span length,  $M_{ae}$  is the aeroelastic moment per unit of  
 203 span length,  $\rho$  is the fluid density,  $U$  is the flow speed,  $K = (B\omega)/U$  is the reduced frequency,  $B$  is  
 204 the deck width,  $\omega$  the circular frequency of oscillation,  $h$  is the heave displacement,  $\alpha$  is the  
 205 torsional rotation,  $\dot{h}$  and  $\dot{\alpha}$  are the time derivatives and  $H_i^*$  and  $A_i^*$  ( $i = 1, \dots, 4$ ) are the flutter  
 206 derivatives.

207 Assuming prescribed harmonic forced oscillations  $h = h_0 e^{i\omega t}$  and  $\alpha = \alpha_0 e^{i\omega t}$ , where  $h_0$  and  $\alpha_0$   
 208 are the amplitudes of the oscillations, and also that motion-induced forces are linear functions of  
 209 the movement; after some manipulation, the following expressions are obtained for the  
 210 identification of the flutter derivatives:

$$H_1^* = -\left(\frac{U}{Bf}\right)^2 \frac{C_l \sin \varphi_{L-h}}{(2\pi)^2 h_0/B} \quad (7.a)$$

$$A_1^* = -\left(\frac{U}{Bf}\right)^2 \frac{C_m \sin \varphi_{M-h}}{(2\pi)^2 h_0/B} \quad (7.e)$$

$$H_2^* = -\left(\frac{U}{Bf}\right)^2 \frac{C_l \sin \varphi_{L-\alpha}}{(2\pi)^2 \alpha_0} \quad (7.b)$$

$$A_2^* = -\left(\frac{U}{Bf}\right)^2 \frac{C_m \sin \varphi_{M-\alpha}}{(2\pi)^2 \alpha_0} \quad (7.f)$$

$$H_3^* = \left(\frac{U}{Bf}\right)^2 \frac{C_l \cos \varphi_{L-\alpha}}{(2\pi)^2 \alpha_0} \quad (7.c)$$

$$A_3^* = \left(\frac{U}{Bf}\right)^2 \frac{C_m \cos \varphi_{M-\alpha}}{(2\pi)^2 \alpha_0} \quad (7.g)$$

$$H_4^* = \left(\frac{U}{Bf}\right)^2 \frac{C_l \cos \varphi_{L-h}}{(2\pi)^2 h_0/B} \quad (7.d)$$

$$A_4^* = \left(\frac{U}{Bf}\right)^2 \frac{C_m \cos \varphi_{M-h}}{(2\pi)^2 h_0/B} \quad (7.h)$$

211

212 where  $\varphi_{L-h}$ ,  $\varphi_{L-\alpha}$ ,  $\varphi_{M-h}$  and  $\varphi_{M-\alpha}$  are the phase lags of the fluctuating aeroelastic lift and  
 213 moment with respect to the heave and pitch harmonic oscillations and  $C_l$  and  $C_m$  are the amplitudes  
 214 of the non-dimensional aeroelastic lift and moment.

215 It must be borne in mind that in Larsen and Walther (1998), whose results are used later for  
 216 validation, the flutter derivatives are computed dividing equations (7.a) to (7.h.) by 2.

#### 217 4. RELATIONSHIPS BETWEEN FLUTTER DERIVATIVES

218 As mentioned in the introduction, a number of publications can be found in the literature reporting  
 219 several relationships amongst flutter derivatives. Tubino (2005) has derived the following  
 220 relationships between heave-related and pitch-related flutter derivatives assuming the linear  
 221 formulation hypothesis for the self-excited forces:

$$H_1^*(K) = KH_3^*(K) - \frac{C_d}{K} \quad (8.a)$$

$$A_1^*(K) = KA_3^*(K) \quad (8.c)$$

$$H_4^*(K) = -KH_2^*(K) \quad (8.b)$$

$$A_4^*(K) = -KA_2^*(K) \quad (8.d)$$

222 The above equations are similar to the ones reported by Matsumoto (1996) apart from the  $(H_1^*, H_3^*)$   
223 relationship, that does not consider the term containing the drag coefficient. For streamlined  
224 sections with low drag coefficient its contribution is nearly negligible.

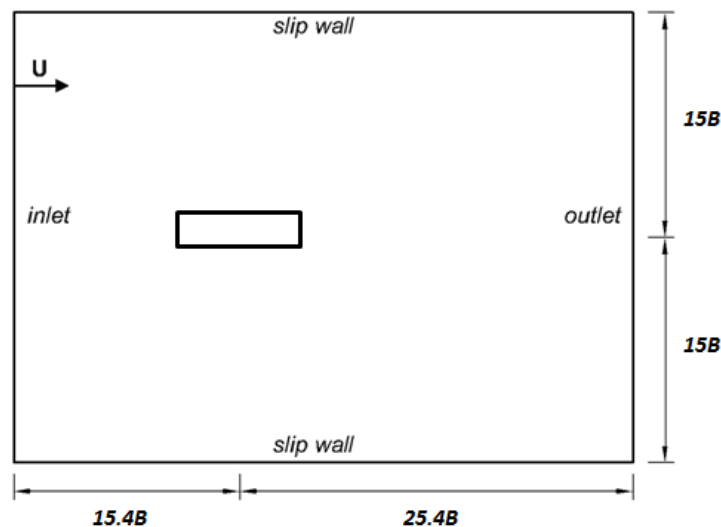
225 Experimental validation of the former relationships reported in Tubino (2005), has shown that the  
226 relationships between  $(H_1^*, H_3^*)$  and  $(A_1^*, A_3^*)$  were satisfied for all the cases considered while the  
227 relationships between  $(H_2^*, H_4^*)$  and  $(A_2^*, A_4^*)$  were closely verified for streamlined deck cross-  
228 sections, since minor discrepancies are identified between experimental realizations and the  
229 approximated values. In Matsumoto (1996), the reported relationships between flutter derivatives  
230 are confirmed for rectangular cylinders less affected by vortex generation, proposing as a reference  
231 lower bound a 5:1 ratio.

## 232 5. GEOMETRY AND COMPUTER MODELING

233 Two different geometries have been considered as case studies in the present work: a  $B/H=4.9$   
234 rectangular cylinder ( $H$  is the section depth or height), and the generic G1 deck section, described  
235 in Larsen and Walther (1998), representative of streamlined box decks.

### 236 5.1. $B/H=4.9$ rectangular cylinder

237 Figure 1 shows the layout of the flow domain and boundary conditions employed in the rectangular  
238 cylinder simulations. The flow domain considered for the rectangular cylinder case is  $40.8B$  by  $30B$   
239 similar to the size employed in successful simulations by other researchers such as Fransos and  
240 Bruno (2010).



241

242 Figure 1. Flow domain definition and boundary conditions for the  $B/H=4.9$  rectangular cylinder  
243 (not to scale).

244 A constant velocity inlet has been set at the upwind boundary (the left side in the figure) of the  
245 computational domain. The incoming flow has a turbulence intensity of 1 % along with a  $0.1B$   
246 turbulent length scale as per Ribeiro (2011). A pressure outlet at atmospheric pressure has been  
247 imposed at the right side (see figure 1). The upper and lower boundaries have been defined as slip  
248 walls. The corners of the prism have been modeled as sharp and its walls are defined as non-slip.  
249 When the rectangular cylinder is forced to oscillate the resultant velocity field around the

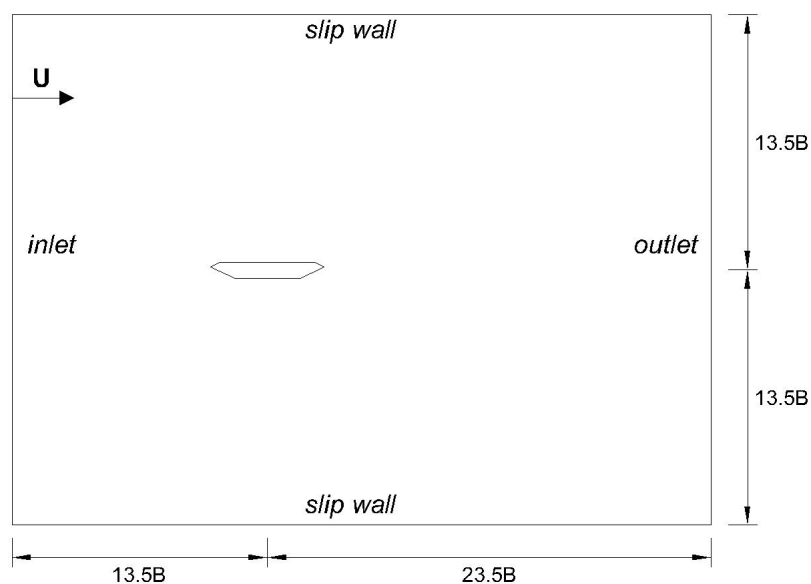
250 rectangular cylinder wall is corrected, so that the velocity of the flow at the moving boundary is  
251 equal to the mesh velocity and therefore no flux across the wall takes place.

252 The numerical schemes adopted in the simulations reported herein are summarized next. The  
253 interpolation of values from the cell centers to face centers is done using a linear scheme. The  
254 gradient terms are discretised using the Gauss scheme with a linear interpolation scheme. For the  
255 divergence terms, the Gauss scheme is also selected, adopting linear upwind and limited linear  
256 interpolation schemes. For the Laplacian terms the choice has been the Gauss scheme with a linear  
257 interpolation scheme and a limited surface normal gradient scheme. The Euler first order bounded  
258 implicit scheme was set for the first time derivative terms.

259 A block structured mesh with a topology similar to the one in Braun and Awruch (2003), has been  
260 generated. The total number of cells is 148320, and the number of cells around the walls of the  
261 rectangular cylinder is 460. For the first layer of cells, the height to width ratio is  $\delta_1/B = 5.11 \times$   
262  $10^{-4}$ , for which the mean value of the non-dimensional height ( $y^+ = (\delta_1 u_*)/\nu$ , where  $\delta_1$  is the  
263 height of the first prismatic grid layer around the deck and  $u_*$  is the friction velocity) is about 1.8  
264 and the maximum value is close to 8 at  $Re = 1.01 \times 10^5$ . These bounds are ~~the~~ similar to those  
265 reported in Sarkic et al. (2012), and in this model, the number of cells with  $y^+ > 4$  is about 5% of  
266 the total number of cells around the rectangular cylinder and they are located mainly in the  
267 windward corners. In both static and forced harmonic oscillations a maximum Courant number of 1  
268 has been imposed, which produces for the static prism at the Reynolds number of reference a mean  
269 non-dimensional time step  $\overline{\Delta s} = \overline{\Delta t}U/B = 6.7 \times 10^{-4}$ . The abundant literature reporting  
270 numerical studies on rectangular cylinders means verification studies concerning mesh size and  
271 time step refinements for the rectangular cylinder case can be avoided, since the authors have used  
272 common mesh topologies and have adopted mesh characteristics and a time step more demanding  
273 than other successful simulations.

## 274 5.2. G1 generic deck cross-section

275 The detailed geometry of the G1 generic cross-section is depicted in figure 8. The flow domain size  
276 in this case is  $37B$  by  $27B$  ( $B$  is the deck width), similar to the size employed in the rectangular  
277 cylinder case. The boundary conditions are the same as in the rectangular cylinder case.



278



279 Figure 2. Flow domain definition and boundary conditions for the G1 section (not to scale).

280 To verify the spatial discretisation, for the streamlined G1 deck section three different grids, with  
281 different mesh densities, have been considered for the static deck case with a  $0^\circ$  angle of attack.  
282 The meshes are identified as Coarse, Medium and Fine grids. In all the cases, a 2D block structured  
283 regular mesh has been used. A high density mesh has been defined around the deck cross-section,  
284 the so-called boundary layer mesh, taking special care in order to obtain maximum values for the  
285 first grid non-dimensional height  $y^+$  below 4, which is a more demanding bound than the one set  
286 by Sarkic et al. (2012) for a similar problem. In this manner, no wall functions are required and the  
287 turbulence model equations are integrated along the viscous sublayer. The thickness of this layer is  
288  $B/25$ . The Coarse mesh comprises 25 rows of elements in this zone and the height of the first  
289 element around the cross-section is defined as  $\delta_1/B = 2.08 \times 10^{-4}$ , while the expansion ratio  
290 between the end cell and the start cell is 25. For the Medium (Figure 3) and Fine meshes the  
291 boundary layer definition was identical: 50 rows considering an expansion ratio of 10, which gives  
292 a first cell non-dimensional height  $\delta_1/B = 2.03 \times 10^{-4}$ , very close to the Coarse mesh case in  
293 order to be able to drive conclusions from the verification analyses since the  $y^+$  values are  
294 comparable for the three cases. For a Reynolds number  $Re = 1.07 \times 10^5$ , these mesh arrangements  
295 offer a mean value of the  $y^+$  around the deck close to 1, with a very limited number of cells with  
296  $y^+ > 2$  located at the windward corners of the deck. The maximum value of  $y^+$  for the three cases  
297 is about 3.7.

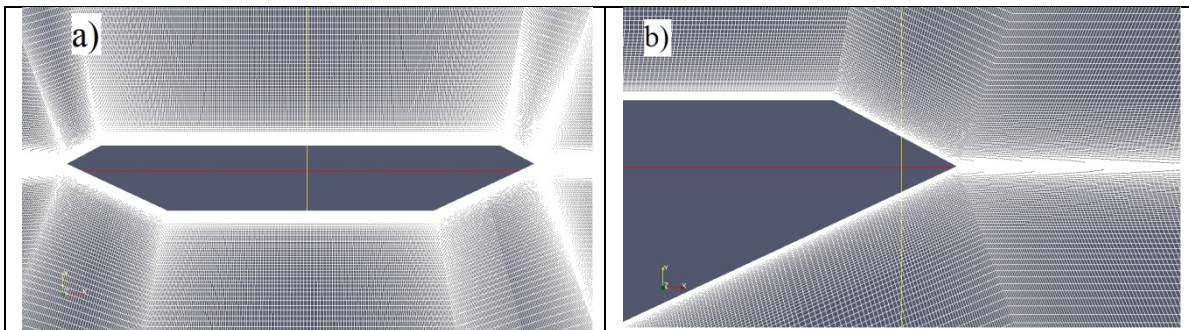
298 In table 1 the total number of cells, the number of cells around the deck section and the integral  
299 aerodynamic parameters are reported along with the standard deviation (prime symbol) values of  
300 the force coefficients for each mesh.

Table 1. Properties and results of the grid-refinement study for the G1 section.

Grid	Total cells	Cells around deck	$S_t$	$C_d$	$C_l$	$C_m$	$C'_d$	$C'_l$	$C'_m$
Coarse	149600	640	0.20	0.056	-0.026	0.035	0.0003	0.010	0.0022
Medium	268150	770	0.19	0.057	-0.033	0.034	0.0006	0.022	0.0047
Fine	363300	770	0.19	0.057	-0.034	0.034	0.0006	0.022	0.0047

301

302



303 Figure 3. G1 section block structured grid: a) close-up of the deck and b) detail around the lee-ward  
304 corner of the deck.

305 The main discrepancies found have been the lower values in the standard deviation of the force  
306 coefficients and the slight underestimation of the lift coefficient when the coarse mesh has been

307 used. Consequently the Coarse mesh has been disregarded and Medium mesh is adopted hereafter  
 308 since the results are similar to the ones obtained using the Fine mesh at a lower computational cost.

309 Regarding the analysis of the sensitivity of the solution depending on the chosen time step, two  
 310 different maximum Courant numbers equal to 1 and 0.5 have been considered in order to check the  
 311 influence of the temporal discretisation (Mannini et al., 2010). In table 2, where the non-  
 312 dimensional time step is defined as  $\overline{\Delta s} = \overline{\Delta t}U/B$ , the numerical results obtained are reported,  
 313 finding that they offer very close figures; therefore the higher maximum Courant number is  
 314 retained for the remaining simulations.

Table 2. Results of the time-refinement study for the G1 section.

Max. Co. numb.	$\overline{\Delta s}$	$S_t$	$C_d$	$C_l$	$C_m$	$C'_d$	$C'_l$	$C'_m$
1	3.5e-4	0.19	0.057	-0.033	0.034	0.0006	0.022	0.0047
0.5	1.8e-4	0.20	0.058	-0.037	0.034	0.0006	0.019	0.0040

315

### 316 5.3. Grid movement strategy

317 The computer implementation of the ALE formulation requires a mesh-update method that assigns  
 318 mesh-node velocities or displacements at each calculation time step (Donea et al., 2004).

319 In the simulations conducted in this research the boundary motion is defined by the prescribed  
 320 forced oscillations of the bluff body, which follows a sinusoidal law with given frequency and  
 321 amplitude. On the other hand, the exterior boundaries of the fluid domain are fixed along the  
 322 simulations. The whole mesh is allowed to deform between the moving and fixed boundaries.

323 Amongst the available mesh movement algorithms a Laplacian smoothing technique for each  
 324 component of the node-mesh position has been chosen (Oliver, 2009). According to Jasak and  
 325 Rusche (2009), the Laplace equation can be expressed as:

$$\nabla \cdot k \nabla \mathbf{u} = 0 \quad (9)$$

326 where  $\mathbf{u}$  is the node-mesh displacement vector and  $k$  is the diffusion coefficient.

327 In this work the mesh control is achieved by computing the motion of the grid points solving the  
 328 Laplace equation with variable diffusivity using a method based on the quadratic inverse distance  
 329 from the oscillating boundary. This prevents the distortion of the smallest elements around the  
 330 rectangular cylinder (Löhner, 2008).

### 331 5.4. Forced oscillations characteristics and application of relationships between flutter 332 derivatives

333 With the aim of limiting the computational cost of obtaining the set of 8 flutter derivatives, the  
 334 relationships between flutter derivatives (8.a–8.d) reported in Tubino (2005) are applied. As a  
 335 consequence, only half of the simulations are required, which represents a substantial reduction in  
 336 the computational demands of the problem. The pitch degree of freedom has been chosen as the  
 337 one for carrying out the numerical simulations; therefore the  $H_2^*$ ,  $H_3^*$ ,  $A_2^*$  and  $A_3^*$  flutter derivatives  
 338 are computed by means of the CFD simulations, while the  $H_1^*$ ,  $H_4^*$ ,  $A_1^*$  and  $A_4^*$  flutter derivatives are  
 339 estimated using equations (8.a) to (8.d). The amplitude of the forced oscillations in the present  
 340 work is  $\alpha_0 = 1^\circ$  for the two considered application examples. The sign convention adopted herein

341 has been the same as in Sarkar et al. (2009): heave and aeroelastic lift force positive downward,  
 342 while the aeroelastic moment and rotation have been considered positive for a nose-up rotation.

343

## 344 6. RESULTS AND DISCUSSION

### 345 6.1 $B/H=4.9$ rectangular cylinder

#### 346 6.1.1 Flow simulation around the static $B/H=4.9$ rectangular cylinder

347 In table 3 the Strouhal number, the mean drag coefficient and the standard deviation of the lift and  
 348 drag coefficients at  $Re = 1.01 \times 10^5$  are presented along with experimental data from Schewe  
 349 (2009) and the numerical data computed using two different 2D URANS approaches. The URANS  
 350 references which have been considered for comparison are: Ribeiro (2011) who reports, amongst  
 351 others, the results of a Reynolds Stress Model (RSM) simulation and Mannini et al. (2011) where  
 352 the Linearised Explicit Algebraic (LEA) version of the Explicit Algebraic Reynolds Stress Model  
 353 (EARSM) coupled with the standard  $k-\omega$  turbulence model is employed. It must be borne in mind  
 354 that in the references used for validation the ratio of the rectangular cylinder is  $B/H=5$ . In table 3,  
 355 the reference dimension for drag coefficient and the standard deviations is  $B$ , therefore the data in  
 356 Mannini et al. (2011), Ribeiro (2011) and Schewe (2009) which are based on  $H$ , have been  
 357 modified for comparison. For the simulation of the  $B/H=4.9$  static rectangular cylinder the  
 358 simulated length has been about 100 non-dimensional time units and the reported results in table 3  
 359 have been averaged along a non-dimensional time  $s = tU/B = 74$ .

Table 3.  $B/H=4.9$  rectangular cylinder: Strouhal number and force coefficients.

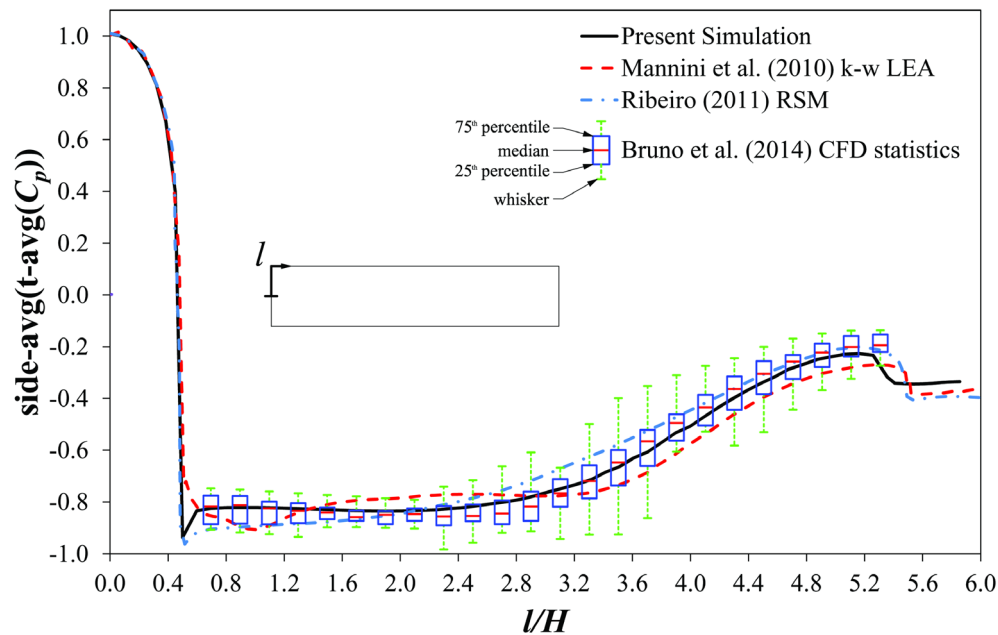
	$S_f$	$C_d$	$C'_d$	$C'_l$
Present simulation	0.123	0.227	0.0049	0.193
Mannini et al. (2011) – LEA $k-\omega$	0.094	0.212	0.0038	0.215
Ribeiro (2011) - RSM	0.073	0.234		0.18
Schewe, (2009) – EXP.	0.111	0.206		$\approx 0.08$

360

361 Table 3 shows a good agreement with the experimental and numerical data, particularly taking into  
 362 account that, since the aspect ratio of the rectangular cylinder considered in the simulation is lower  
 363 than 5, it must show slightly higher values for both Strouhal number and drag force coefficients  
 364 according with the trend in drag coefficient and Strouhal number for rectangular cylinders with  
 365 aspect ratios between 4 and 6, reported in Shimada and Ishihara (2012). It is notable how Menter's  
 366  $k-\omega$  SST turbulence model considered in this simulation offers results comparable with the  
 367 sophisticated LEA approach in Mannini et al. (2011). The proximity of the Strouhal number in this  
 368 simulation to the experimental value obtained in Schewe (2009) should also be highlighted and  
 369 therefore a better prediction of this parameter than in Ribeiro (2011) has been obtained.

370 As a further validation of the reported simulations, in figure 4 the side-averaged (between the upper  
 371 and lower half perimeters) and time-averaged distribution of the pressure coefficient  $C_p$  of the  
 372 static ratio  $B/H=4.9$  rectangular cylinder are reported along with the results in Mannini et al. (2010)  
 373 for the  $k-\omega$  LEA turbulence model, Ribeiro (2011) for the RSM and the statistics for the CFD  
 374 realizations reported in Bruno et al. (2014). The side-averaged and time-averaged pressure  
 375 coefficient of the ratio 4.9 rectangular cylinder is very close to the median values on the long side  
 376 of the rectangular cylinder ( $l/H$  between 0.7 and 5.3, being  $l$  the length along the half of the  
 377 perimeter of the rectangular cylinder, as it is described in figure 4) which indicates that the  
 378 accuracy of the simulation is comparable with the CFD realizations in the frame of the BARC

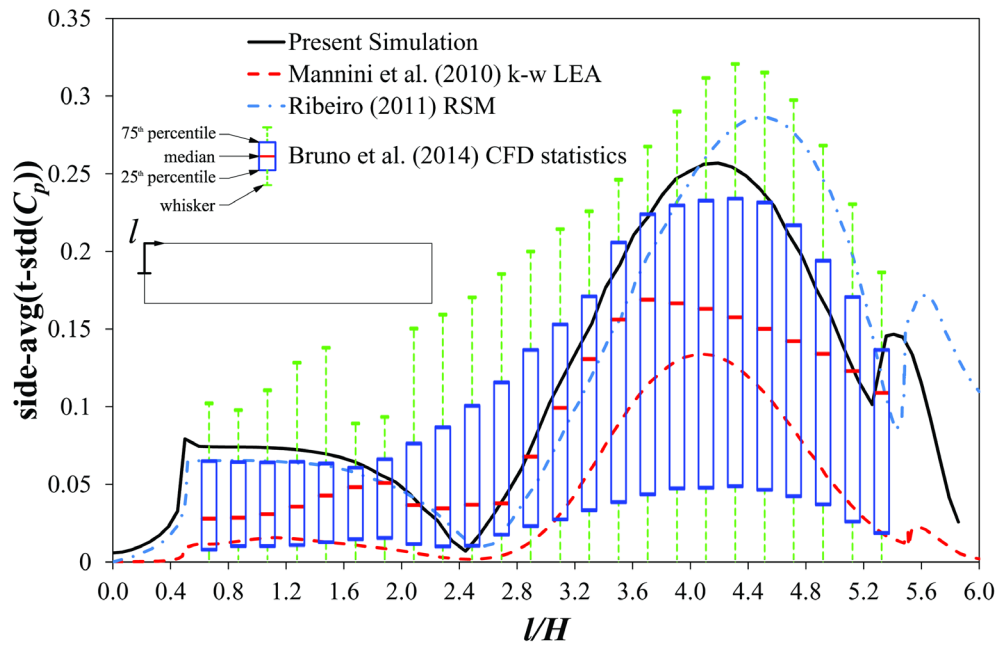
379 initiative. Furthermore, the numerical results correctly reproduce the experimental data for the 5:1  
 380 rectangular cylinder, bearing in mind the scatter in the wind tunnel tests available in the literature.



381

382 Figure 4. Side-averaged and time-averaged  $C_p$  distributions around  $B/H=4.9$  and  $B/H=5$   
 383 rectangular cylinders.

384 In figure 5, the side-averaged distribution of the standard deviation ~~in time~~ of the pressure  
 385 coefficient is reported along with the statistical data for the CFD realizations in Bruno et al. (2014)  
 386 and the simulations in Mannini et al. (2010) and Ribeiro (2011). In Bruno et al. (2014) the scatter  
 387 in the distribution of the standard deviation of the pressure coefficient has been shown for both  
 388 experimental and numerical realizations. The standard deviation distribution of the  $C_p$  reported for  
 389 the  $B/H=4.9$  rectangular cylinder is well inside the boundaries of the BARC realizations and it is  
 390 particularly close to the RSM simulation in Ribeiro (2011). It has reported in Bruno et al. (2014)  
 391 that RANS simulations present a minimum in the standard deviation of the pressure coefficient at  
 392 about  $2H$  from the windward corner. This minimum is also present in the simulation reported in  
 393 this work.



394

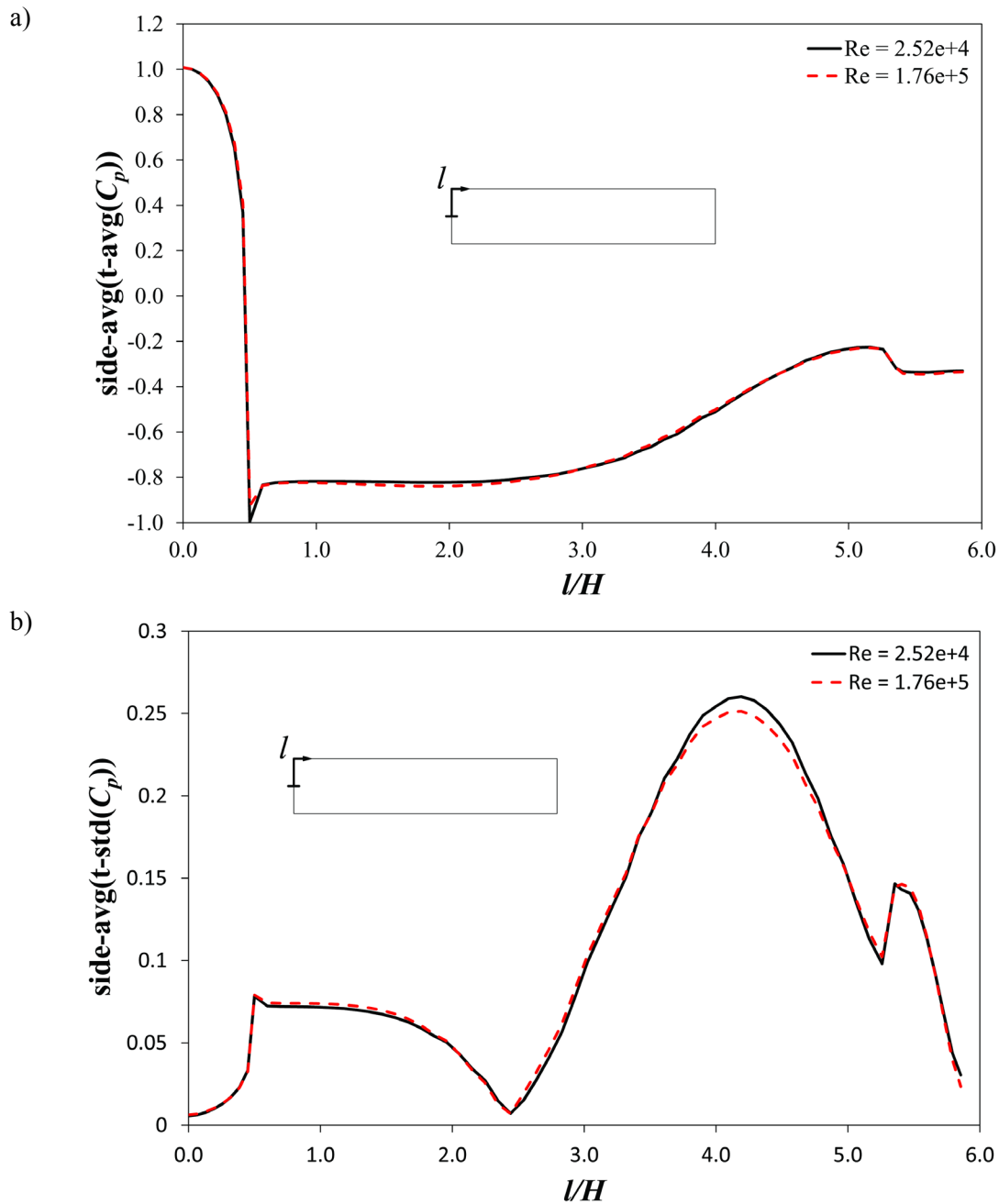
395 Figure 5. Side-averaged distributions around  $B/H=4.9$  and  $B/H=5$  rectangular cylinders of the  
 396 standard deviation in time of  $C_p$ .

397 Based on the comparison of the drag coefficient, the standard deviation of the lift coefficient, the  
 398 Strouhal number and the distribution to the time-averaged and time-standard deviation of the  
 399 pressure coefficient, the agreement of the present simulation with the experimental and numerical  
 400 data in the literature can be considered adequate.

#### 401 6.1.2 Flutter derivatives of the $B/H=4.9$ rectangular cylinder

402 The flutter derivatives for the aspect ratio 4.9 rectangular cylinder have been computed over a  
 403 range of reduced velocities  $U_R = U/(f \cdot B) = (0.88, 26.40)$ . In order to cover the whole range of  
 404 reduced velocities, three frequencies of oscillation have been considered (0.5 Hz., 1 Hz. and 3 Hz.)  
 405 in conjunction with flow speeds between 1 m/s and 7 m/s, which means that the range of covered  
 406 Reynolds number is between  $2.52 \times 10^4$  and  $1.76 \times 10^5$ . In some cases ( $U_R = 2.6, 5.3, 10.6$  and  $15.84$ ),  
 407 the same reduced velocity has been computed with different combinations of flow velocity and  
 408 frequency of oscillation in order to verify the independence of the results with the combination of  
 409 both parameters.

410 Since the same mesh has been retained for all the simulations, the non-dimensional height  $y^+$   
 411 reaches a maximum value close to 11 for the maximum Reynolds number ( $Re = 1.76 \times 10^5$ ;  $U = 7$   
 412 m/s), while the mean value of  $y^+$  is about 2.7. For the minimum Reynolds number ( $Re =$   
 413  $2.52 \times 10^4$ ;  $U = 1$  m/s), the maximum  $y^+$  reaches a value close to 3.5 and the mean value of  $y^+$  is  
 414 0.6. With the aim of ascertaining the effect of the differences in the  $y^+$  numbers on the simulations  
 415 at the lower and upper bounds of the Reynolds number, as well as the dependency of the  
 416 aerodynamic characteristics with the Reynolds number, the side-averaged and time-averaged along  
 417 with the side-averaged time-standard deviation distributions of the pressure coefficient are  
 418 presented for  $U = 1$  and  $U = 7$  m/s (Figure 6).

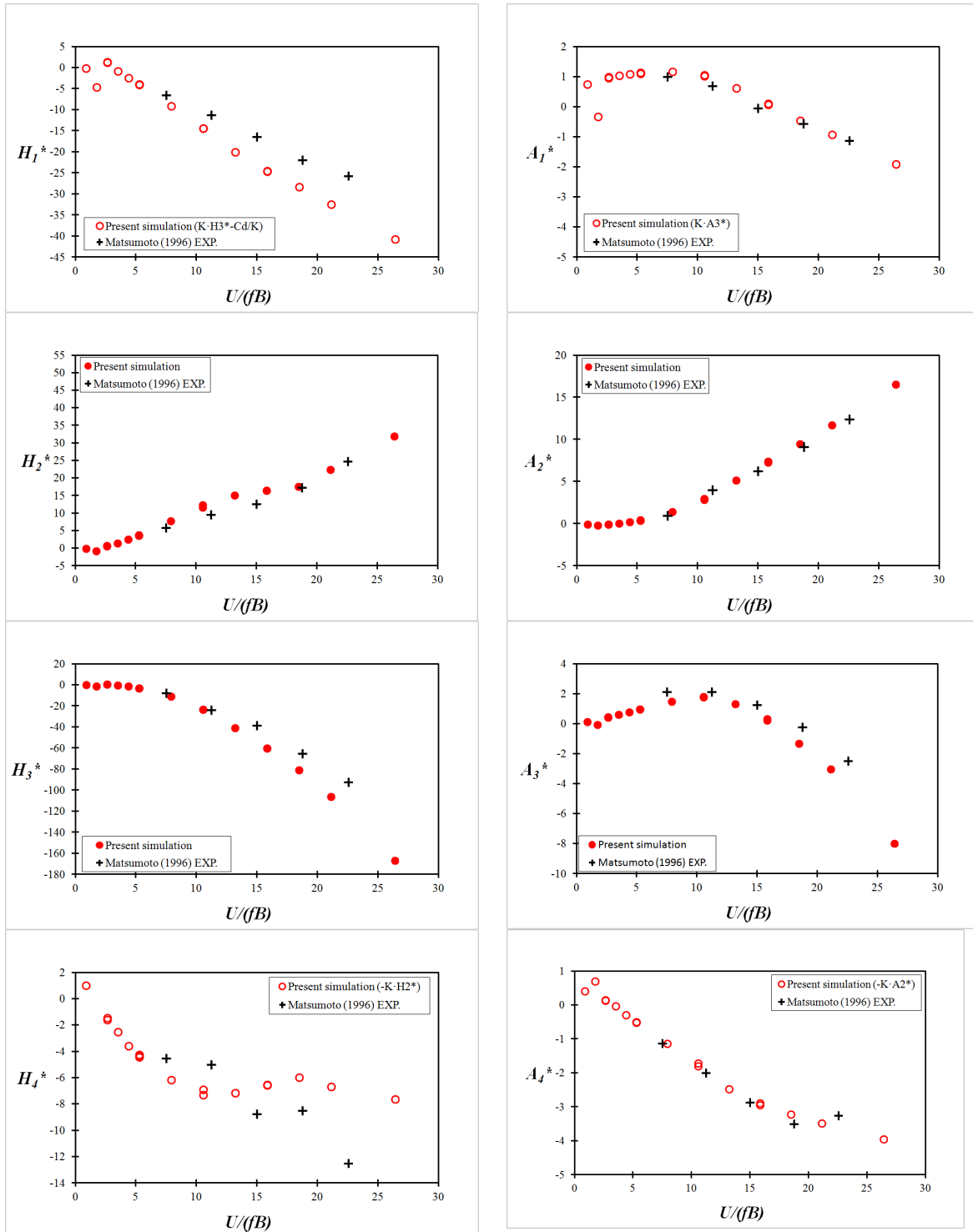


419 Figure 6. Side-averaged distributions around  $B/H=4.9$  rectangular cylinder of the a) time-averaged  
 420 and b) time-standard deviation of  $C_p$  for  $U = 1$  and  $U = 7$  m/s.

421 Figure 6 shows similar results for the side-averaged distributions of the time-averaged and the  
 422 standard deviation of the pressure coefficient. Only small differences in the peak value of the  
 423 distribution of the standard deviation of the pressure coefficient around the rectangular prism can  
 424 be identified. Consequently, the relatively high values of the maximum  $y^+$  at  $U = 7$  m/s do not  
 425 jeopardize the accuracy of the simulation. At the same time, the aerodynamic characteristics of the  
 426 static sharp edged rectangular cylinder at  $0^\circ$  angle of attack seems to be quite insensitive to the  
 427 Reynolds number, as it has been reported in Holmes (2007), citing Scruton (1981). Besides this, in  
 428 the set of reduced velocities considered for the computation of the flutter derivatives, the maximum  
 429 flow speed of 7 m/s is adopted for a single reduced velocity  $U_R = 18.48$ . In the same manner, the  
 430 flow speed of 6 m/s is employed only for repeated values of  $U_R = 5.3$  and  $U_R = 15.84$ . Therefore,

431 in the set of flutter derivatives which are presented next, the majority of the simulations have been  
 432 conducted at  $Re \leq 1.26 \times 10^5$ .

433 In figure 7 the flutter derivatives computed from these simulations are reported along with the  
 434 experimental data in Matsumoto (1996). The length of the simulations reported in the following has  
 435 been between 40 and 260 non-dimensional time units, depending on the flow speed and the  
 436 frequency of oscillation.



437 Figure 7. Flutter derivatives of the  $B/H=4.9$  rectangular cylinder: computed flutter derivatives and  
 438 comparison with experimental data in Matsumoto (1996).

439 The estimated flutter derivatives agree well with the experimental data and only the  $H_4^*$  flutter  
 440 derivative shows some discrepancies with the wind tunnel values. These differences in  $H_4^*$  are  
 441 comparable with the ones found in CFD simulations where forced oscillations in the heave degree  
 442 of freedom have been conducted, such as in Sarwar et al. (2008) for a  $B/H=20$  rectangular cylinder  
 443 or Huang (2009). There are no significant differences for the repeated simulations at the same  
 444 reduced velocities, which points out the relative independence of the results with the various  
 445 combinations of flow speed and frequency of oscillation.

## 446 6.2 G1 generic deck cross-section

### 447 6.2.1 Flow simulation around the static G1 section

448 The drag coefficient, the root mean square of the lift coefficient time history and the Strouhal  
 449 number of the G1 section for  $0^\circ$  angle of incidence computed in this study are compared in table 4  
 450 with the numerical results reported in Larsen and Walther (1998) who applied the Discrete Vortex  
 451 Method in their simulations. In this case the numerical simulation of the static G1 section has been  
 452 extended along 65 non-dimensional time units. The time statistics have been obtained from the  
 453 final 45 non-dimensional time units.

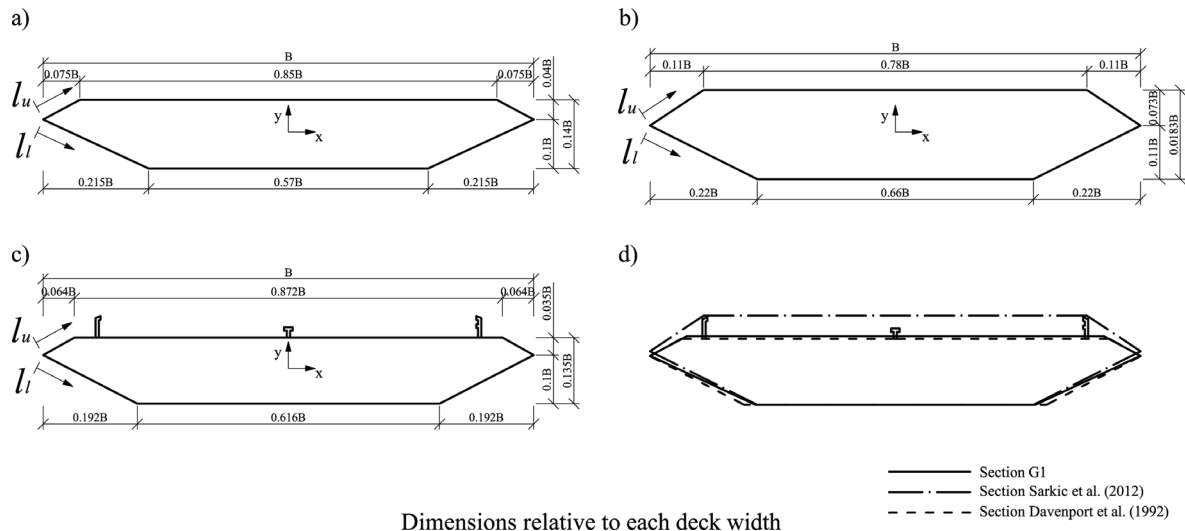
Table 4. Static G1 section: drag coefficient, RMS of the lift coefficient and Strouhal number.

	$C_d$	$C_l^{RMS}$	$S_t$
Present simulation	0.06	0.04	0.19
Larsen and Walther (1998)	0.08	0.07	0.17

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455 The agreement amongst the results for the  $0^\circ$  angle of attack is reasonable, however as a further  
 456 validation of the numerical approach chosen by the authors, the time-averaged pressure coefficient  
 457 distribution along the deck is going to be presented and compared with the experimental data  
 458 reported in Sarkic et al. (2012), where the time-averaged pressure coefficient distribution along a  
 459 bare box deck is provided. For further comparison, the experimental data in Bruno and Khris  
 460 (2003) (taken from Larose, 1992) of the smooth flow tests of a taut strip model of the Great Belt  
 461 Bridge fitted with barriers, has also been included. The geometry of the deck and the position of the  
 462 pressure probes in the aforementioned reference are taken from Davenport et al. (1992). The  
 463 distribution of the ~~time~~ standard deviation of the pressure coefficient is not reported since the  
 464 unsteadiness of the flow was rather weak, providing values of the pressure coefficient standard  
 465 deviation well below the available experimental data, particularly on the windward half of the cross  
 466 section. A similar behavior is described in Sarkic et al. (2012). In figure 8, the geometry of the  
 467 bridge decks considered for validation is described, while in figure 9 the time-averaged pressure  
 468 coefficient distribution is shown.





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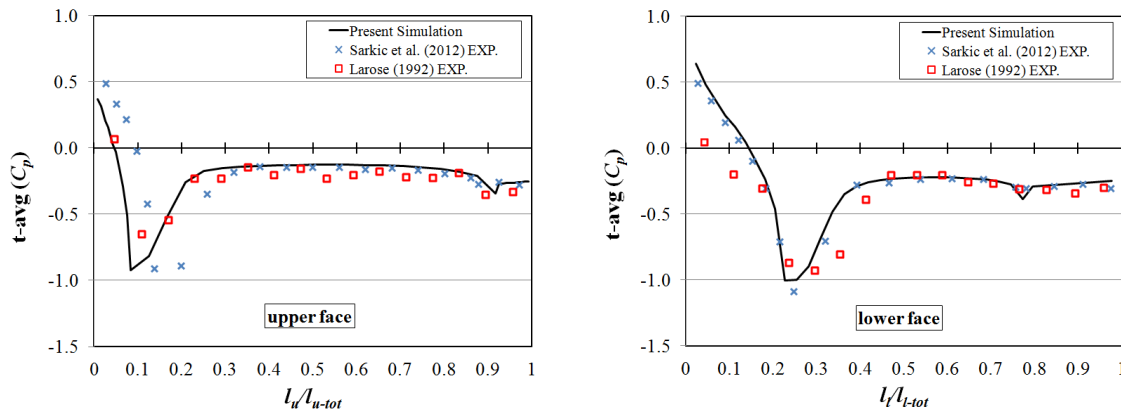
Dimensions relative to each deck width

470

Figure 8. Geometry: a) G1 section b) section in Sarkic et al. (2012) c) section in Davenport et al.

471

(1992) d) comparison between sections.



472

Figure 9. Time-averaged pressure coefficient distribution: numerical results and comparison with experimental data in Sarkic et al. (2012) and Larose (1992).

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The agreement in the pressure coefficient distribution between the numerical simulation and the wind tunnel data in Sarkic et al. (2012) is good. On the upper face, the peak values at the windward corner are correctly simulated and the lateral shift is due to the differences in the geometry in the upper surface (see figure 8). Also the mean pressure distribution along the horizontal and the leeward plates have been accurately obtained. The agreement is even better on the lower surface, since the geometry of the two sections is nearly identical. In the authors' opinion the similitude in the Reynolds number ( $Re \approx 1 \times 10^5$ ) of the numerical simulation and the wind tunnel test has contributed to this close agreement.

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When the numerical results are compared with the wind tunnel data from Larose (1992), some discrepancies can be identified, which can arguably be related to the difference in the Reynolds number of the wind tunnel tests ( $Re = 7 \times 10^4$ ) as well as the presence of the barriers in the tested model. Besides this, discrepancy in the moderate suction on the windward surface in the lower side of the deck has already been commented in Bruno and Khris (2003).

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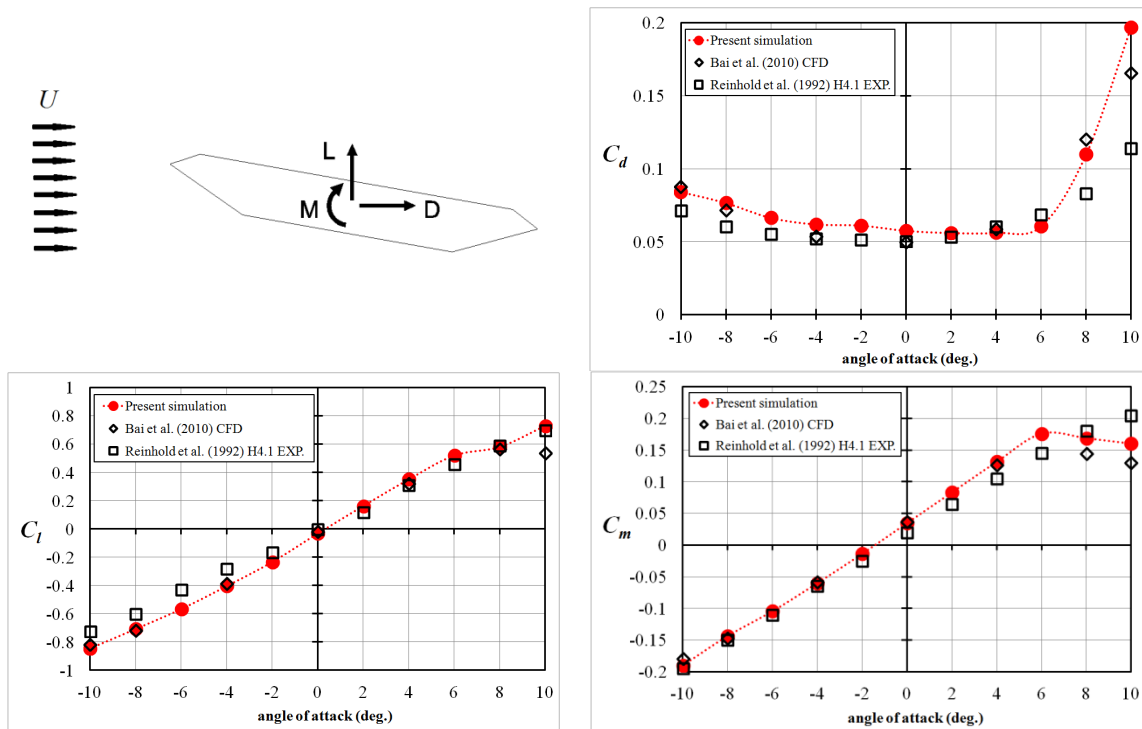
In order to provide a more complete view of the aerodynamic characteristics of the static G1 cross section, the force coefficients in the range of angles of attack ( $-10^\circ$ ,  $10^\circ$ ) are computed with an interval of  $2^\circ$ . The results are compared with the experimental data reported in Reinhold et al.

488

489

490 (1992) for the H4.1 section of the Great Belt Bridge design studies and the 2D numerical results  
 491 published in Bai et al. (2010), for the G1 section.

492 Figure 10 shows the force coefficients of the G1 section. A very good agreement has been obtained  
 493 between the computational results and the experimental data for the similar geometry of the H4.1  
 494 box deck section. In fact, the change in the slope of the moment coefficient for angles of incidence  
 495 higher than  $6^\circ$  has been correctly captured as well as the step increment in the drag coefficient  
 496 for angles of attack higher than  $6^\circ$ . The accuracy of the slopes in the vicinity of  $0^\circ$  for both lift and  
 497 moment coefficients should also be noted.



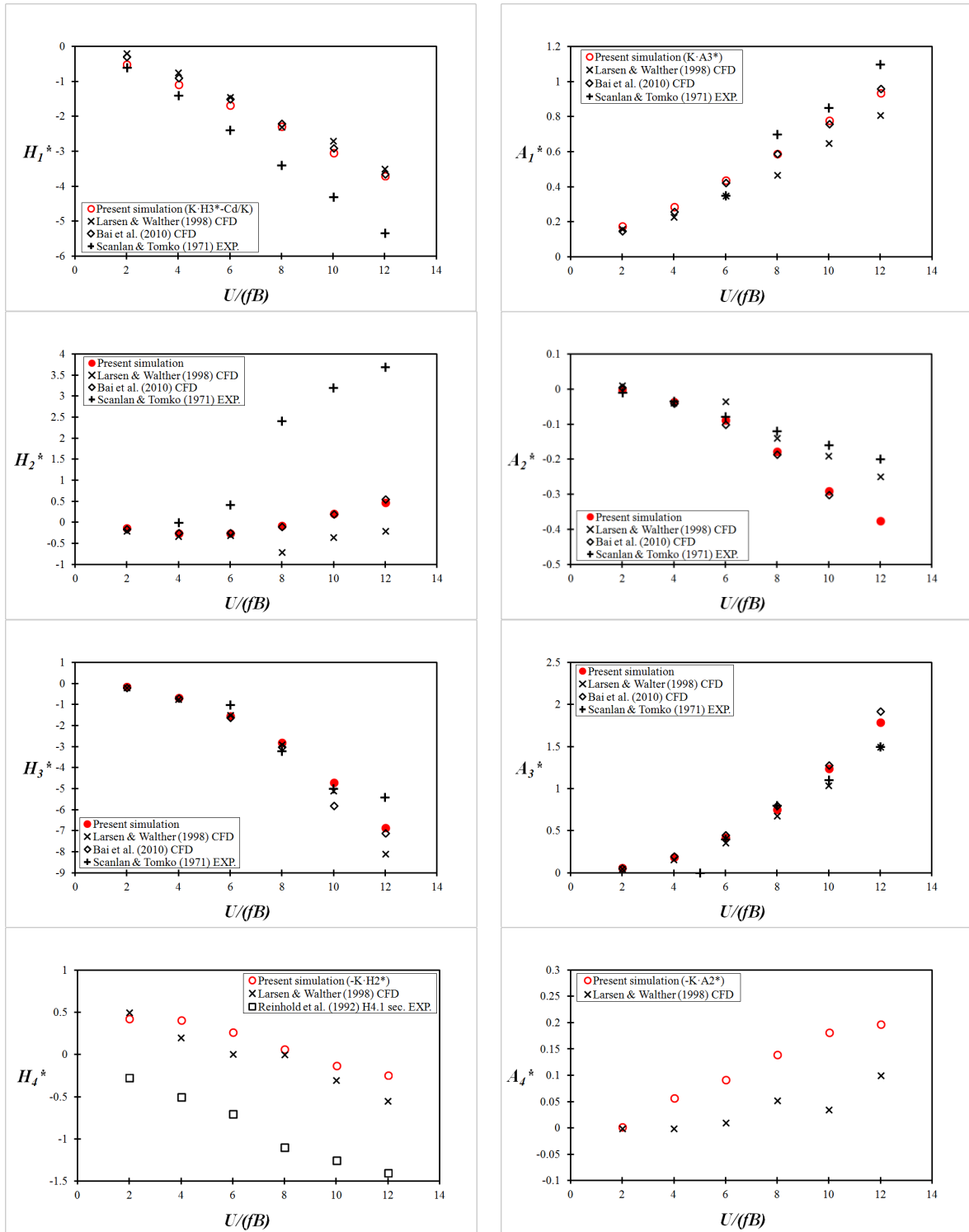
498 Figure 10. G1 section force coefficients: numerical results and comparison with experimental  
 499 (Reinhold et al., 1992) and other numerical data (Bai et al., 2010).

## 500 6.2.2 Flutter derivatives of the G1 section

501 In order to identify by means of a computational approach the flutter derivatives of the G1 generic  
 502 section, forced oscillation simulations were carried out at reduced velocities  $U/(fB)$  equal to 2, 4,  
 503 6, 8 10 and 12, as in Larsen and Walther (1998). Also, the formulae applied for identifying the  
 504 flutter derivatives are the ones reported in Larsen and Walter (1998) and Bai et al (2010), therefore  
 505 the expressions in equations (7.a) to (7h) are divided by 2. The same procedure as in the  
 506 rectangular cylinder case has been applied for decreasing the computational cost. As a  
 507 consequence, instead of 12 computer simulations, only 6 are required, one for each reduced  
 508 velocity considered. In this case the flow velocity is the same in all the simulations and the  
 509 frequency of oscillation is modified in the range (0.833, 5) Hz in order to obtain the reduced  
 510 velocities of interest. The solution for the fixed G1 section has been set as the initial condition for  
 511 the forced oscillation simulations. Since this allows shortening the initial transient, the  
 512 computations have been extended for about 50 non-dimensional time units. For the highest value of  
 513 the reduced velocity,  $U_R=12$ , four complete oscillation periods have been simulated, which is  
 514 greater than the 2.5 periods span adopted in Larsen and Walther (1998).

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In figure 11 the numerical results obtained for  $H_i^*$  and  $A_i^*$  ( $i = 1, \dots, 3$ ) are compared with the experimental ones reported in Scanlan and Tomko (1971). The numerical results obtained by Larsen and Walther (1998), and Bai et al. (2010) for the same deck section are also included in the charts. Since no experimental results are available for the  $H_4^*$  and  $A_4^*$  flutter derivatives of the G1 cross-section, the results for the  $H_4^*$  flutter derivative of the H4.1 section in Reinhold et al (1992) are provided. No experimental data for the  $A_4^*$  flutter derivative of the H4.1 section are available in the literature to the authors' knowledge.



522 Figure 11. Flutter derivatives of the G1 generic section: numerical results and comparison with  
 523 experimental (Scanlan and Tomko, 1971; Reinhold et al., 1992) and numerical (Larsen and  
 524 Walther, 1998; Bai et al., 2010) data.

525 A very good agreement has been found for the flutter derivatives related to the pitch forced  
 526 oscillation:  $H_3^*$ ,  $A_2^*$  and  $A_3^*$ , which have been obtained from the numerical simulations. For the  $H_2^*$   
 527 flutter derivative, similar discrepancies as in Bai et al. (2010) have been obtained. In fact, for this  
 528 flutter derivative, in the case of box decks, differences between experimental data and CFD based  
 529 evaluations can be found in other references in the literature, such as Jeong and Kwon (2003), Zhu  
 530 et al. (2007), Ge and Xiang (2008) or Brusiani et al. (2013). For the approximated heave-related  
 531 flutter derivatives  $H_1^*$  and  $A_1^*$  the obtained results agree with wind tunnel test data and their  
 532 accuracy is comparable with the other CFD-based simulations. For the flutter derivatives  $H_4^*$   
 533 and  $A_4^*$  it is more difficult to properly assess the reliability of the approximated values since  
 534 experimental data are not available. It has been found that for the  $H_4^*$  flutter derivative the present  
 535 simulation provides values very similar to those reported by Larsen and Walther (1998). In the  
 536 same manner, the slope is almost the same as for the H4.1 experimental flutter derivative and the  
 537 upwards shift of the numerical results can also be found, for instance, in Brusiani et al. (2013)  
 538 where the flutter derivatives of the H4.1 section were specifically computed. For the  $A_4^*$  flutter  
 539 derivative the approximated values do not show important differences in value with respect to the  
 540 ones in Larsen and Walther (1998).

541 In order to assess the degree of accuracy in the simulations reported in this work, in table 5 the  
 542 relative errors in the value of the flutter derivatives  $H_1^*$ ,  $H_2^*$ ,  $H_3^*$ ,  $A_1^*$ ,  $A_2^*$  and  $A_3^*$ , for which  
 543 experimental data are available, are reported. It must be borne in mind that the data for the lower  
 544 reduced velocities cannot be identified from the charts in Scanlan and Tomko (1971) for some of  
 545 the flutter derivatives.

546 The relative errors of the numerical values taking as reference the experimental values are  
 547 evaluated according to the following formula:

$$e = \frac{|exp. value - num. value|}{|exp. value|} \quad (9)$$

548

549 Table 5. Relative errors in the evaluation of the flutter derivatives of the G1 section

Flutter derivative	$U_R$	Present simulation	Larsen and Walther (1998)	Bai et al. (2010)
$H_1^*$	2	0.14	0.67	0.50
	4	0.23	0.46	0.36
	6	0.30	0.40	0.38
	8	0.33	0.32	0.35
	10	0.29	0.37	0.33
	12	0.31	0.34	0.32
$H_2^*$	6	1.60	1.71	1.57
	8	1.03	1.29	1.04
	10	0.93	1.11	0.94
	12	0.87	1.05	0.85
$H_3^*$	6	0.54	0.50	0.60

	8	0.12	0.09	0.06
	10	0.06	0.02	0.16
	12	0.27	0.50	0.31
$A_1^*$	6	0.25	0.00	0.21
	8	0.16	0.33	0.16
	10	0.08	0.24	0.11
	12	0.15	0.26	0.13
$A_2^*$	2	0.95	2.00	1.50
	4	0.03	0.14	0.14
	6	0.09	0.56	0.25
	8	0.47	0.17	0.54
	10	0.81	0.19	0.88
	12	0.88	0.25	
$A_3^*$	6	0.04	0.10	0.13
	8	0.06	0.15	0.00
	10	0.13	0.05	0.16
	12	0.19	0.00	0.28

550

551 From table 5, it can be concluded that the accuracy of the three simulations is equivalent, being the  
552 median of the relative errors 0.26 in the present simulation, and 0.33 and 0.32 in Larsen and  
553 Walther (1998) and Bai et al. (2010). In this respect, it is notable how the approximated values  
554 obtained using the proposed approach for the  $H_1^*$  and  $A_1^*$  flutter derivatives are comparable with the  
555 values reported in Larsen and Walther (1998) and Bai et al. (2010) where the harmonic oscillations  
556 in the heave degree of freedom were explicitly computed.

## 557 7. CONCLUDING REMARKS

558 In this article, the force coefficients and the flutter derivatives of an aspect ratio 4.9 rectangular  
559 cylinder and a streamlined deck type G1 cross-section have been computed based on a 2D URANS  
560 approach, applying Menter's  $k-\omega$  SST turbulence model. A block structured mesh has been used  
561 and the open source CFD solver OpenFOAM has been applied.

562 The static response of the rectangular cylinder at a  $0^\circ$  angle of attack has agreed well with the  
563 experimental data in Schewe (2009), the RSM simulation in Ribeiro (2011) and sophisticated 2D  
564 numerical simulations where the Boussinesq assumption is substituted by an EARSM approach  
565 (Mannini et al., 2011).

566 For the G1 section, the influence of the spatial and temporal discretisations in the numerical results  
567 has been studied. Since both experimental and numerical results of the force coefficients and flutter  
568 derivatives are available in the literature for this particular cross-section, the current computational  
569 results have been validated against the experimental ones and also the accuracy of the simulations  
570 reported herein can be compared with CFD results published by other researchers.

571 The distribution of the time-averaged pressure coefficient around the G1 section agrees well with  
572 experimental data available in the literature for similar geometries. The force coefficients of the  
573 deck cross-section for angles of attack in the range  $-10^\circ$  and  $+10^\circ$  have been obtained. It has been

574 found that they are in good agreement with the experimental and numerical data in Reinhold et al.  
575 (1992) and Bai et al. (2010).

576 A notable contribution of this work has been the application of the existing formulae relating the  
577 flutter derivatives (Tubino, 2005) in a CFD based approach. This has allowed the computer  
578 demands of this burdensome problem to be reduced. The pitch-related flutter derivatives have been  
579 extracted from the pitch forced oscillation simulations while the heave-related ones have been  
580 estimated using the expressions in the literature. For the two cases studied a very good agreement  
581 with the experimental flutter derivatives has been found, and at least comparable accuracy with  
582 other numerical simulations where both pitch and heave forced oscillations had been numerically  
583 computed.

584 This work can be considered a step forward towards the routine use of CFD based techniques in the  
585 aerodynamic and aeroelastic design of long span bridges since it has been demonstrated the  
586 adequacy of the computational results using an efficient 2D approach. Furthermore it ~~but~~ is also a  
587 step forward in the application of numerical optimization techniques in the shape design of bridges,  
588 for which efficient, reliable and computational non-cumbersome CFD techniques are a must. In this  
589 respect, a fully computational approach for the evaluation of force coefficients and flutter  
590 derivatives, as the one reported herein, is required for the application of numerical optimization  
591 techniques.

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