

A non-intrusive method to compute water discharge in pipes with a low depth-to-diameter ratio using ultrasonic Doppler velocimetry

Authors: I. Fraga¹, L. Cea², J. Puertas³, J. Anta⁴

¹ GEAMA (Environmental and Water Engineering Group), E.T.S Caminos, Universidade de A Coruña, Campus Elviña s/n, 15071 A Coruña, Spain. Corresponding autor. email: ignacio.fraga@udc.es

² GEAMA (Environmental and Water Engineering Group), E.T.S Caminos, Universidade de A Coruña, Campus Elviña s/n, 15071 A Coruña, Spain. email: luis.cea@udc.es

³ GEAMA (Environmental and Water Engineering Group), E.T.S Caminos, Universidade de A Coruña, Campus Elviña s/n, 15071 A Coruña, Spain. email: jpuertas@udc.es

⁴ GEAMA (Environmental and Water Engineering Group), E.T.S Caminos, Universidade de A Coruña, Campus Elviña s/n, 15071 A Coruña, Spain. email: janta@udc.es

Abstract

A non-intrusive method to calculate the water depth and discharge in partially full pipes using data from a single ultrasonic Doppler velocimeter (UDV) profiler is presented. The position of the free surface is identified from the velocity profiles measured with the UDV. The flow discharge is computed from an approximated parameterization of the velocity field in the cross section, using a single measured velocity profile. The proposed methodology was applied to steady and unsteady flow conditions in two different pipes with diameters of 90 and 200 mm, and depth-to-diameter ratios up to 0.35. Under these conditions, the water depth and discharge were measured with mean absolute errors of the order of 1mm and 0.1 l/s in the 90 mm pipe, and 0.5 mm and 0.05 l/s in the 200 mm pipe. These errors are almost independent of the discharge.

Keywords: ultrasonic Doppler velocimetry; flow discharge measurement; non-intrusive measurement; partially full pipes; urban drainage

Introduction

An accurate determination of the flow rate in partially full pipes is as a major necessity but also a difficult task. Conventional flow metering methods (such as area-velocity or turbine flow meters) present important limitations as they usually require strict conditions to obtain accurate results, such as long straight pipes and high water depths, among others (Mori et al. 2001). Because of these restrictions, the use of ultrasonic Doppler velocimetry has become increasingly popular in measuring velocity profiles and water discharges in pipes. In this context, the main advantage of an ultrasonic Doppler velocimeter (UDV) profiler is its capacity to measure in a non-intrusive way velocity profiles, with the data rate being virtually independent of the seeding concentration of particles in the water. A detailed description of

34 ultrasonic Doppler velocimetry techniques can be found in Takeda (1990, 1995, 1999 and 2012) and
35 Lemmin and Rolland (1997).

36 The capabilities of the UDV have resulted in many advances and studies in recent years, most of
37 which have focused on pressurized flows. Mori et al. (2001) and Wada et al. (2004) developed methods to
38 compute the flow rate in pressurized pipes integrating the instantaneous velocity profiles measured with
39 the UDV, with relative errors in the computed discharge below 1%. In free surface flows, most of the
40 research concerns on the interaction between air bubbles and water (Suzuki et al. 2002; Murai et al. 2006)
41 and its effects on the UDV velocity measurements (Longo 2006)

42 In this technical note, we propose a methodology to compute the water discharge in partially full
43 pipes with low depth-to-diameter ratios using a single UDV profiler, which measures the velocity of the
44 fluid at several sampling volumes in an axial profile. The aim is to evaluate the possibility of using a
45 single UDV profiler as a non-intrusive discharge-measuring device in partially full pipes, and to quantify
46 the accuracy of the discharge measures. The position of the free surface is identified using the velocity
47 profiles measured with the UDV, and the discharge is computed from a simplified parameterization of the
48 velocity field in the cross section. The methodology was calibrated and validated using experimental data
49 obtained in the laboratory in two pipes of diameters 90 and 200 mm.

50 **Experimental setup**

51 The experimental setup consists of the pipeline shown in Fig. 1. Water is pumped from a tank (1), flows
52 through a valve (2), and it is discharged into a manhole (3). From the manhole, water flows into pipes 1
53 and 2 and discharges into a cylindrical basin (6) with a pressure sensor (7). The characteristics of the
54 pipes are detailed in Table 1.

55 **Table 1.** Characteristics of the pipes used in the experimental setup.

56 The water discharge at the line outlet is evaluated from the time variation of the water volume in
57 the cylindrical basin where the pipe spills. The water depth at the basin is measured with a pressure sensor
58 at a frequency of 1 Hz.

59 Four meters downstream of the manhole in pipe 1, and at the central point of pipe 2, small
60 orifices are opened on the top of the pipe to install ultrasonic distance sensors with a clamp-on system,
61 pointing toward the bottom of the pipes to measure the water depth. The recording frequency of these
62 sensors is 2 Hz. At the same position where the distance sensors are located, two DOP2000 (Signal

63 Processing S.A.) UDVs are secured with brackets to the bottom of the pipes pointing at the center of each
64 pipe. The angle between the probes and the pipe longitudinal axis (α in Fig. 2) is 65° . As pointed in
65 Yokoyama et al. (2004), small inaccuracies in the angle of the UDV transducer can be the main cause of
66 error when computing discharges from measured velocities. In the results presented in this work, a
67 deviation of only 1° implies an error of approximately 5% on the computed discharge. An accurate setup
68 of the probe angle is therefore of great importance.

69 The transducers are in contact with the pipe wall, and the gap between the probe and the wall is
70 filled with a gel (AquaGel 100, Parker Laboratories, S.A.) that works as a coupling medium to allow the
71 propagation of the ultrasonic waves. The angles between the ultrasonic beam and the pipe longitudinal
72 axis in both the pipe wall and the liquid (θ and β in Fig. 2) are computed from the refraction law taking
73 into account the sound celerity of the pipe wall, the ultrasonic gel and the liquid.

74 **Methodology**

75 ***Data treatment***

76 *Despiking*

77 Raw data from the UDV contains corrupt information that needs to be filtered, mainly because of the
78 Doppler noise and the aliasing of the signal. Several studies have been published in which different
79 despiking techniques are proposed and compared (Cea et al. 2007; Jesson et al. 2013). In the present
80 study, the filter proposed by Goring and Nikora (2002) was used to detect and remove spikes from the
81 raw velocity data registered with the UDV transducers.

82 *Velocity projection*

83 The UDV gives the velocity component in the beam axis direction. Measured velocities are projected in
84 the pipe longitudinal direction, assuming that the water flows parallel to the pipe axis.

85 *Distance correction*

86 The following correction proposed in Wang et al. (2003) is applied to the raw distances measured with
87 the UDV to take into account the different sound celerity in the pipe wall, the ultrasonic gel and the
88 liquid. The correction introduces an offset in the position of the sampling volumes due to the different
89 celerity of the ultrasonic waves in the ultrasonic gel and the pipe wall. The real distance traveled by the
90 ultrasonic beam is computed as:

$$d = d_g + d_w + \left(\frac{d^*}{c_L \sin \alpha} - \frac{d_g}{c_g \sin \alpha} - \frac{d_w}{c_w \sin \beta} \right) c_L \sin \theta \quad (1)$$

91 where c_w , c_g , and c_L are the sound celerities in the pipe wall, the ultrasonic gel, and the liquid respectively,
 92 d is the real distance traveled by the ultrasonic beam, d^* is the raw distance measured by the UDV and d_g
 93 and d_w are the distances traveled by the ultrasonic waves along the ultrasonic gel and the pipe wall (Fig.
 94 2).

95 *Measurement volume correction*

96 The correction proposed in Nowak (2002) is used at the sampling volumes in contact with the pipe wall
 97 and with the free surface. This correction takes into account that at the interface between medias (liquid –
 98 air and liquid – pipe wall) only part of the sampling volume is located inside the fluid. The correction
 99 consists in assigning the measured velocity to the centroid of only the volume located inside the fluid
 100 instead of the mass center of the whole sampling volume.

101 *Position of the free surface*

102 The position of the free surface is determined from the velocity profiles measured with the UDV. To
 103 calibrate the methodology, the water depth was additionally measured with the ultrasonic distance
 104 sensors.

105 The velocity profile measured with the UDV has a local minimum just above the maximum
 106 velocity in the profile (Fig. 3). The position of this minimum, referred to as d_{UDV} in Table 2, is in close
 107 agreement with the position of the free surface measured with the distance sensor (d_M in Table 2).
 108 Therefore, a simple criterion to evaluate the water depth from the UDV measurements is to locate the free
 109 surface at the position of this minimum. The ratio between the water depth corresponding to the
 110 maximum velocity and d_{UDV} is similar in all the profiles registered, with values of approximately 0.85
 111 and 0.9 in pipes 1 and 2, respectively.

112 **Table 2.** Water depths measured with the ultrasonic distance sensor (d_M), obtained from the UDV
 113 profiles (d_{UDV}), and computed with the Sobel filter (d_s). Φ_{int} is the interior diameter of the pipe.

114 The water depths obtained with this criterion are compared in Table 2 with those computed with
 115 the method described in Murai et al. (2006), in which the free surface is identified using the Sobel filter
 116 on the velocity profiles. The Sobel filter overestimates in all cases the water depth, the mean absolute

117 error (MAE) being 2.65 and 1.09 mm in pipes 1 and 2, respectively. With the proposed criterion, the
 118 MAE is reduced to 0.94 and 0.47 mm in pipes 1 and 2, respectively.

119 ***Parameterization of the velocity field***

120 To compute the discharge in a pipe cross section, it is necessary to integrate the velocity field in the wet
 121 section. Since in partially full pipes the flow is not axisymmetric, a parameterization of the velocity field
 122 is needed to estimate the velocity distribution from a single profile. Most of the existing methods to
 123 estimate the velocity distribution in pipes and open channels are based on probabilistic and entropy-
 124 maximization approaches, such as the ones described in Marini et al. (2011), Chiu (1988) and Chiu and
 125 Hsu (2006). These methods assume that the discharge and the mean velocity in the cross section are
 126 known and therefore, they cannot be applied to evaluate the discharge from a single velocity profile.

127 In this study, we propose a parameterization of the velocity distribution given by a series of
 128 isovelocity curves defined from each sampling volume of the UDV in the following way. If the velocity
 129 in a sampling volume is lower than the velocity at the free surface, the corresponding isovelocity curve is
 130 defined as an arc with the same center as the pipe cross section and a radius defined by Eq. (2). This is the
 131 case of the isovelocity curve 1 in Fig. 4. In the sampling volumes in which the velocity is higher than the
 132 velocity at the free surface, the isovelocity curve is defined by an arc concentric to the pipe cross section
 133 and a chord parallel to the free surface. This is the case of the isovelocity curve 2 in Fig. 4, which is
 134 defined by Eq. (2) and (3). It should be remarked that this parameterization is just an approximation of the
 135 real velocity distribution in partially full pipes with a low depth-to-diameter ratio, the aim being to
 136 evaluate the discharge and not to reproduce the exact velocity field. The advantages of the proposed
 137 parameterization are its simplicity, that it does not rely on any calibration parameter and that it gives quite
 138 accurate discharge estimations, as will be shown in the following sections.

$$R_1 = R_{pipe} - d_1 \quad \varphi = 2 \cdot \arccos \left(\frac{R_{pipe} - d_{UDV}}{R_{pipe} - d_1} \right) \quad (2)$$

$$R_2 = R_{pipe} - d_{2A} \quad \varphi = 2 \cdot \arccos \left(\frac{R_{pipe} - d_{2B}}{R_{pipe} - d_{2A}} \right) \quad (3)$$

139 From the velocity parameterization given by Eq. (2) and (3), the discharge is computed as

$$Q = \sum_i^{n-1} (A_{i+1} - A_i) \left(\frac{V_{i+1} + V_i}{2} \right) \quad (4)$$

140 where A_i is the area inside the i^{th} isovelocity curve, V_i is the velocity of the i^{th} isovelocity curve, and n is
141 the number of isovelocity curves, which is equal to the number of sampling points of the central UDV.

142 ***Results under steady-state conditions***

143 The previous methodology was calibrated under steady conditions for the discharges shown in Table 3. In
144 all cases, the velocity profiles were measured for 20 s with a measuring frequency of 10 Hz, resulting in
145 200 profiles per discharge. In both pipes, the mean absolute relative error (MARE) on the computed
146 discharges is below 5%, with a slightly better performance in pipe 1.

147 **Table 3.** Discharges computed with the proposed parameterization. Relative errors are shown in
148 parentheses.

149 In the results presented in Table 3, if the correction proposed by Wang et al. (2003) is not
150 applied the computed discharge decreases. For the lowest discharge this decrease is almost 2% and 7% in
151 pipes 1 and 2 respectively, while the effect of the correction nearly halves for the highest discharge. On
152 the contrary, if the correction proposed by Nowak (2002) is ignored, the computed discharges increase
153 approximately 8% in both pipes for the lowest discharge. The increase is reduced to 1.8 % and 0.4 % in
154 pipes 1 and 2 respectively for the highest discharges. It is interesting to notice that when both of the
155 corrections are considered the impact on the computed discharge is reduced since they have the opposite
156 effect.

157 **Validation under unsteady conditions**

158 The proposed methodology was validated under unsteady conditions with a discharge increasing from
159 zero to 2 l/s in 130 s and then decreasing again to zero (Fig. 5). The maximum depth-to-diameter ratios
160 achieved during the validation were 0.35 and 0.14 in pipes 1 and 2, respectively. Velocity profiles were
161 measured with a sampling frequency of 10 Hz and averaged over 1 s in order to evaluate the outlet
162 hydrograph with a frequency of 1 Hz.

163 The outlet hydrograph computed from the UDV data is compared against the discharges
164 measured at the pipe line outlet in Fig. 5. The global agreement is very satisfactory, especially in pipe 2.
165 Differences between computed and measured discharges alternate positive and negative values in both
166 pipes with no noticeable bias (the mean errors on the discharge are 0.0180 and 0.0002 l/s in pipes 1 and 2,
167 respectively). It is also interesting to notice that there is no significant trend in the absolute error as the

168 discharge increases, which implies that the relative error diminishes as the discharge increases. The mean
169 errors in the computed discharge during the whole experiment are shown in Table 4.

170 **Table 4.** Mean errors and standard deviation of the error in the computed discharge.

171 **Conclusions**

172 A methodology to compute the water depth and flow rate in partially full pipes with a low depth-to-
173 diameter ratio using data from a single ultrasonic Doppler velocimeter profiler was presented. The
174 methodology was tested under steady and unsteady conditions in two pipes of 90 and 200 mm diameters.
175 Discharge and water depth ranged up to 2.5 l/s and 31 mm, with depth-to-diameter ratios up to 0.35.
176 Absolute errors on the water depth are below 1 and 0.5 mm in the 90 and 200 mm pipes respectively.
177 Regarding the water discharge, errors are higher in the 90 mm pipe, where they reach values of the order
178 of 0.1 l/s, while in pipe 2 errors nearly halve. No clear trend was observed between the accuracy of the
179 methodology and the flow rate.

180 Although the proposed methodology has only been tested in 90 and 200 mm diameter pipes, it
181 might be applicable to larger pipes. However, its application to hydraulic conditions different from the
182 ones presented in this paper, especially in terms of the water depth-to-diameter ratio, may need a different
183 parameterization of the velocity field in the cross section to ensure that the discharge is properly
184 computed.

185 **References**

- 186 Cea, L., Puertas, J., and Pena, L. (2007). "Velocity measurements on highly turbulent free surface flow
187 using ADV." *Exp. Fluids*. 42, 333-348.
- 188 Chiu, C.L. (1988). "Entropy and 2-D velocity distribution in open channels." *J. Hydraul. Eng.* 114, 738-
189 756.
- 190 Chiu, C.L., and Hsu, S.M. (2006). "Probabilistic approach to modeling of velocity distributions in fluid
191 flows." *J. Hydrol.* 316, 28-42.
- 192 Goring, D.G., and Nikora, V.I. (2002). "Despiking acoustic Doppler velocimeter data." *J. Hydraul. Eng.*
193 128, 117-126
- 194 Jesson, M., Sterling, M., and Bridgeman J. (2013). "Despiking velocity time-series—Optimisation
195 through the combination of spike detection and replacement methods." *Flow Meas. Instrum.* 30, 45-51.

196 Lemmin, U., and Rolland, T. (1997). "Acoustic velocity profiler for laboratory and field studies." *J.*
197 *Hydraul. Eng.* 123, 1089-1097

198 Longo, S. (2006). "The effects of air bubbles on ultrasound velocity measurements." *Exp. Fluids* 41, 593-
199 602

200 Marini, G., De Martino, G., Fontana, N., Fiorentino, M., and Singh, V.P. (2011). "Entropy approach for
201 2D velocity distribution in open-channel flow." *J. Hydraul. Res.* 49, 784-790

202 Mori, M., Takeda, Y., Taishi, T., Furuichi, N., Aritomi, M., and Kikura, H. (2002). "Development of a
203 novel flow metering system using ultrasonic velocity profile measurement." *Exp. Fluids* 32, 153-160

204 Murai, Y., Fujii, H., Tasaka, Y., and Takeda, Y. (2006). "Turbulent bubbly channel flow investigated by
205 ultrasound velocity profiler." *J. Fluid. Sci. Tech.* 1, 12-23.

206 Nowak, M. (2002). "Wall shear stress measurement in a turbulent pipe flow using ultrasound Doppler
207 velocimetry." *Exp. Fluids* 33, 249-255

208 Suzuki, Y., Nakagawa, M., Aritomi, M., Murakawa, H., Kikura, H., and Mori, M. (2002) "Microstructure
209 of the flow field around a bubble in counter-current bubbly flow." *Exp. Therm. Fluid Sci.* 26, 221-227.

210 Takeda, Y. (1990). "Development of ultrasound velocity profile monitor." *Nucl. Eng. Design* 126, 277-
211 285

212 Takeda, Y. (1995). "Velocity profile measurement by ultrasonic Doppler method." *Exp. Therm. Fluid Sci.*
213 10, 444-453.

214 Takeda, Y. (1999). "Ultrasonic Doppler method or velocity profile measurement in fluid dynamics and
215 fluid engineering." *Exp. Fluids* 26, 177-178

216 Takeda, Y. (2012). "Ultrasonic Doppler velocity profiler for fluid flow." Springer, Japan. doi:
217 10.1007/978-4-431-54026-7

218 Wada, S., Kikura, H., Aritomi, M., Mori, M., and Takeda, Y. (2004). "Development of pulse ultrasonic
219 Doppler method for flow rate measurement in power plant multilines flow rate measurement on metal
220 pipe." *J. Nucl. Sci. Technol.* 41(3), 339-346

221 Wang, T., Wang, J., Ren, F., and Jin, Y. (2003). "Application of Doppler ultrasound velocimetry in
222 multiphase flow." *Chem. Eng. J.* 92(1), 111-122

223 Yokoyama, K., Kashiwaguma, N., Okubo, T., and Takeda, Y. (2004). "Flow measurement in an open
224 channel by UVP". *Proc ISUD4*, 204-210
225

226 **Figure Captions**

227 **Fig. 1.** Schematic representation of the experimental setup.

228 **Fig. 2.** Scheme of the UDV transducers setup

229 **Fig. 3.** Velocity profiles in pipes 1 (left) and 2 (right), for water discharges Q_1 , Q_3 , and Q_5 (**Table 2**).

230 Dashed lines correspond to d_{UDV} .

231 **Fig. 4.** Parameterization of the isovelocity curves (left) from the measured velocity profile with the UDV

232 (right). The dots in the velocity profile represent sampling volumes.

233 **Fig. 5.** Hydrographs in pipes 1 (left) and 2 (right) directly measured and computed from UDV

234 measurements.

235

236 **Tables and table captions**237 **Table 5.** Characteristics of the pipes used in the experimental setup.

Pipe	Exterior diameter (mm)	Pipe wall thickness (mm)	Material	Slope (%)	Length (m)
1	90	2.5	Polypropylene	1.75	5
2	200	4.9	PVC	0.87	6

238

239 **Table 6.** Water depths measured with the ultrasonic distance sensor (d_M), obtained from the UDV
240 profiles (d_{UDV}), and computed with the Sobel filter (d_s). Φ_{int} is the interior diameter of the pipe.

Q (l/s)	Water depths in pipe 1				Water depths in pipe 2			
	d_M (mm)	d_M/Φ_{int} (%)	d_{UDV} (mm)	d_s (mm)	d_M (mm)	d_M/Φ_{int} (%)	d_{UDV} (mm)	d_s (mm)
2.55	28.27	33.3	28.77	29.99	31.33	14.9	30.51	31.39
2.08	23.94	28.2	25.10	27.55	28.52	12.6	28.77	29.64
1.63	21.18	24.9	22.66	25.10	24.67	11.1	24.40	26.15
0.99	18.41	21.7	19.66	20.88	19.30	9.7	20.03	20.91
0.54	15.00	17.6	15.32	16.54	13.64	7.9	13.92	14.80

241

242 **Table 7.** Discharges computed with the proposed parameterization. Relative errors are shown in
243 parentheses.

Q (l/s)	Computed discharges	
	pipe 1 (l/s)	Pipe 2 (l/s)
2.55	2.59 (1.7%)	2.44 (-4.2%)
2.08	2.04 (-2.1%)	2.14 (2.8%)
1.63	1.52 (-6.8%)	1.48 (-9.1%)
0.99	0.97(-2.2%)	0.98 (-1.2%)
0.54	0.56 (3.0%)	0.57 (4.3%)
MARE	3.1%	4.3%

244

245 **Table 8.** Mean errors and standard deviation of the error in the computed discharge.

Pipe	MAE (l/s)	ME (l/s)	Error Standard Deviation (l/s)
1	0.1050	-0.0180	0.1192
2	0.0520	-0.0002	0.0712

246

1-Tank
2-Valve

3-Manhole
4-UDV

5-Distance sensors
6-Outlet basin

7-Pressure sensor









